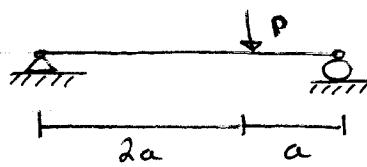


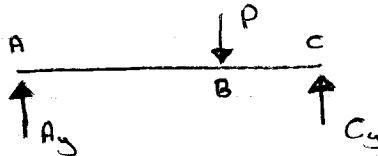
(1)

MARCH, 13/17

Example: Determine the deflection of a simply supported beam. $EI = \text{const.}$



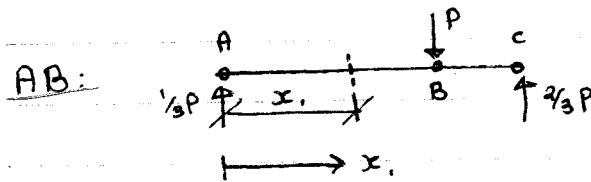
Solution:



$$\sum M_A = 0$$

$$C_y(3a) - P(2a) = 0$$

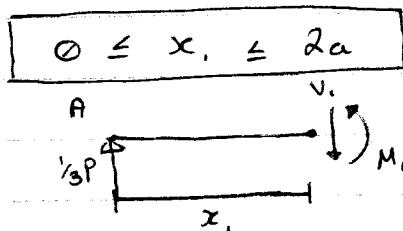
$$C_y = \frac{2}{3}P$$



$$\sum N_c = 0$$

$$P(a) - A_y(3a) = 0$$

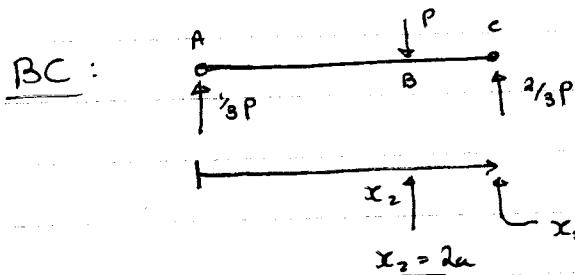
$$A_y = \frac{1}{3}P$$



$$\sum M = 0$$

$$M_1 - \frac{1}{3}Px_1 = 0$$

$$M_1(x_1) = \frac{1}{3}Px_1$$

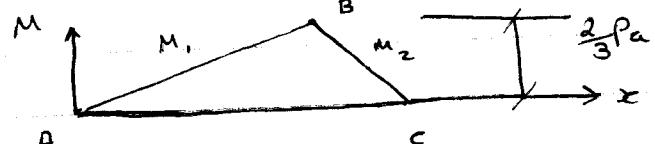
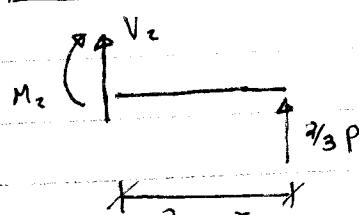


$$\sum M = 0$$

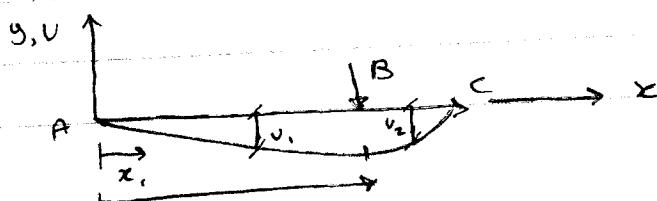
$$-M_2 + \frac{2}{3}P(3a - x_2) = 0$$

$$\therefore M_2(x_2) = \frac{2}{3}P(3a - x_2)$$

$2a \leq x_2 \leq 3a$



Elastic Curve:



(2)

$$[AB] \quad 0 \leq x_1 \leq 2a$$

$$EI \frac{d^2v_1}{dx_1^2} = M_1 = \frac{1}{3}Px_1 \quad (1)$$

$$\text{THEN } \Rightarrow EI \frac{dv_1}{dx_1} = \frac{1}{6}Px_1^2 + C_1 \quad (2)$$

$$\text{THEN } \Rightarrow EI v_1 = \frac{1}{18}Px_1^3 + C_1x_1 + C_2 \quad (3)$$

$$\text{At A, } x_1 = 0, \quad v_1 = 0$$

$$\text{Eq.(2)} \Rightarrow : \quad 0 = 0 + 0 + C_2$$

$$C_2 = 0$$

$$[BC] \quad 2a \leq x_2 \leq 3a$$

$$EI \frac{d^2v_2}{dx_2^2} = M_2 = 2/3P(3a - x_2)$$

$$= -2/3P(x_2 - 3a)$$

$$\Rightarrow EI \frac{dv_2}{dx_2} = -\frac{1}{3}P(x_2 - 3a)^2 + C_3 \quad (4)$$

$$\Rightarrow EI v_2 = -\frac{1}{9}P(x_2 - 3a)^3 + C_3(x_2 - 3a) + C_4 \quad (5)$$

$$z = x_2 - 3a$$

$$\frac{dv_2}{dx_2} = \frac{dv_2}{dz}$$

$$\frac{d^2v_2}{dx_2^2} = \frac{d^2v_2}{dz^2}$$

$$\Rightarrow EI \frac{d^2v_2}{dz^2} = -\frac{2}{3}Pz$$

$$EI \frac{dv_2}{dz} = -\frac{1}{3}Pz^2 + C_3$$

$$EI v_2 = -\frac{1}{9}Pz^3 + C_3z + C_4$$

$$A+C_1 \quad x_2 = 3a, \quad v_2 = 0$$

$$Eq(5): \quad 0 = 0 + 0 + C_4$$

$$C_4 = 0 \quad (6)$$

$$A+B, \quad x_1 = 2a \quad x_2 = 2a$$

$$\underline{AB}: \quad EI v_1 = \frac{1}{18} P(2a)^3 + C_1(2a)$$

$$\underline{BC}: \quad EI v_2 = -\frac{1}{9} P(2a-3a)^3 + C_3(2a-3a)$$

$$(v_1 = v_2) \rightarrow \text{at point } B$$

$$\Rightarrow \frac{1}{18} P(2a)^3 + C_1(2a) = -\frac{1}{9} P(2a-3a)^3 + C_3(2a-3a) \quad (7)$$

$$A+B; \quad x_1 = x_2 = 2a$$

$$\underline{AB}: \quad EI \frac{dv_1}{dx_1} = \frac{1}{6} P(2a)^2 + C_1$$

$$\underline{BC} \quad EI \frac{dv_2}{dx_2} = -\frac{1}{3} P(2a-3a)^2 + C_3$$

$$\boxed{\frac{dv_1}{dx_1} = \frac{dv_2}{dx_2}}$$

$$\Rightarrow \frac{1}{6} P(2a)^2 + C_1 = -\frac{1}{3} (2a-3a)^2 + C_3 \quad (8)$$

$$\text{Solving } (7)(8) \Rightarrow C_1 = -\frac{4}{9} Pa^2 \quad \left. \begin{array}{l} \\ \end{array} \right\} \quad C_3 = \frac{5}{9} Pa^2 \quad (9)$$

Deflection:

$$AB: \quad v_1 = \frac{P}{18EI} (x_1^3 - 8a^2 x_1) \quad 0 \leq x_1 \leq 2a$$

$$BC: \quad v_2 = \frac{P}{9EI} (-x_2^3 + 9ax_2^2 - 22a^2 x_2 + 12a^3) \quad 2a \leq x_2 \leq 3a$$

$$BC: \frac{dV_2}{dx_2} = \frac{P}{9EI} (-3x_2^2 + 18ax_2 - 22a^2) = 0$$

$$x_2 = (3 \pm \sqrt{5/3})a$$

$$x_2 = 4.291a \quad x_2 = 1.709a$$

No solution, max at B or C.

$$At B, x_1 = x_2 = 2a$$

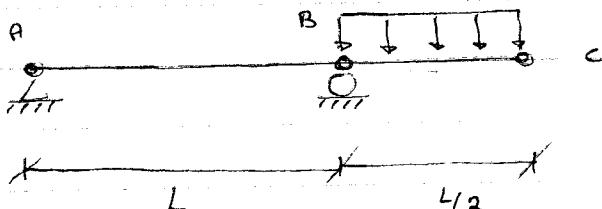
$$V_1 = \frac{P}{18EI} ((2a)^3 - 8a^2 \cdot 2a)$$

$$= \frac{4 Pa^3}{9EI}$$

$$= -0.444 \frac{Pa^3}{EI}$$

$$\therefore V_{\max} = -0.444 \frac{Pa^3}{EI} \quad (x_1 = \sqrt{8/3}a)$$

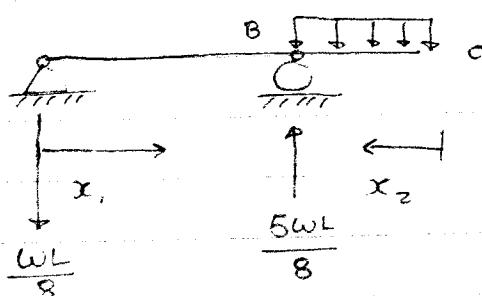
Example:



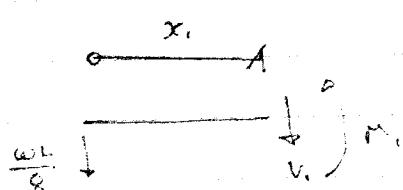
EI = const.

Find the elastic curve.

Solution:



AB $0 \leq x_1 \leq L$



$$M_1 + \frac{WL}{8} x_1 = 0$$

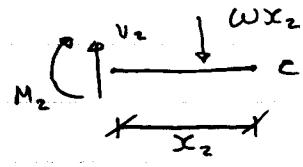
$$M_1 = -\frac{WL}{8} x_1$$

(5)

$$BC: 0 \leq x_2 \leq L/2$$

$$-M_2 - \omega x_2 - \frac{1}{2} x_2 = 0$$

$$M_2 = -\frac{\omega}{2} x_2^2$$



Elastic Curve

AB :

$$EI \frac{d^2v_1}{dx_1^2} = M_1 = -\frac{\omega L}{8} x_1$$

$$EI \frac{dv_1}{dx_1} = -\frac{\omega L}{16} x_1^2 + C_1$$

$$EI v_1 = -\frac{\omega L}{48} x_1^3 + C_1 x_1 + C_2$$

$$BC: EI \frac{d^2v_2}{dx_2^2} = M_2 = -\frac{\omega}{2} x_2^2$$

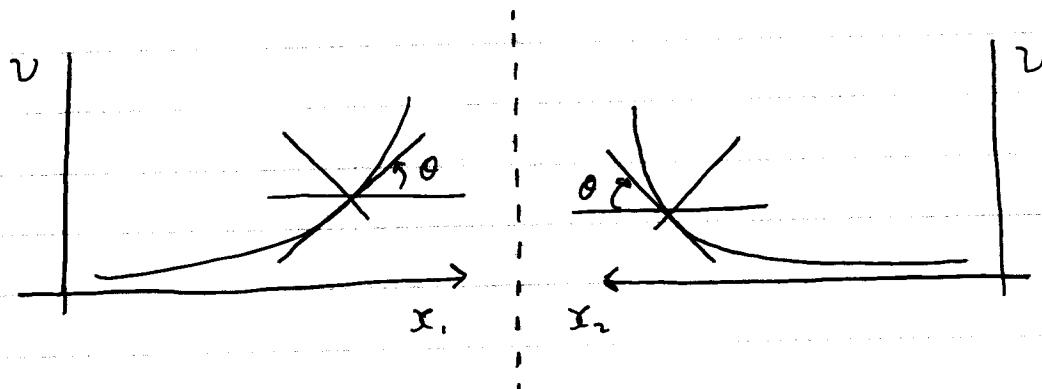
$$EI \frac{dv_2}{dx_2} = -\frac{\omega}{6} x_2^3 + C_3$$

$$EI v_2 = -\frac{\omega}{24} x_2^4 + C_3 x_2 + C_4$$

$$At A, x_1 = 0, v_1 = 0$$

$$At B, x_1 = L, x_2 = L/2$$

$$v_1 = v_2 = 0$$



(6)

$$\text{At } B : \quad x_1 = L \quad x_2 = L/2$$

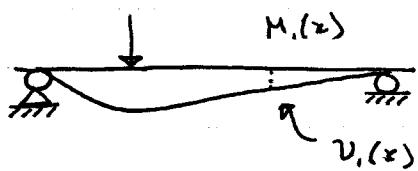
$$\left[\frac{dv_1}{dx_1}(L) = - \frac{dv_2}{dx_2}(L/2) \right]$$

Deflections:

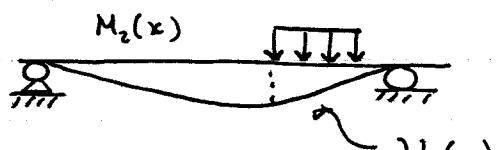
$$\underline{AB} : \quad v_1 = \frac{\omega L}{48EI} \left(-x_1^3 + L^2 x_1 \right)$$

$$\underline{BC} : \quad v_2 = \frac{\omega}{384EI} \left(-16x_2^4 + 24L^3 x_2 - 11L^4 \right)$$

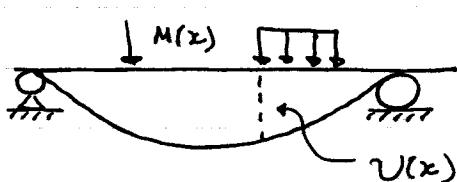
Method of Superposition



$$EI \frac{d^2v}{dx^2} = M_1(x)$$



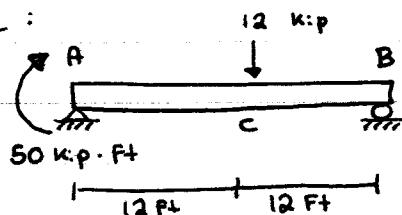
$$EI \frac{d^2v}{dx^2} = M_2(x)$$



$$EI \frac{d^2v}{dx^2} = M(x)$$

$$\begin{aligned} M(x) &= M_1(x) + M_2(x) \\ \Rightarrow v(x) &= v_1(x) + v_2(x) \end{aligned}$$

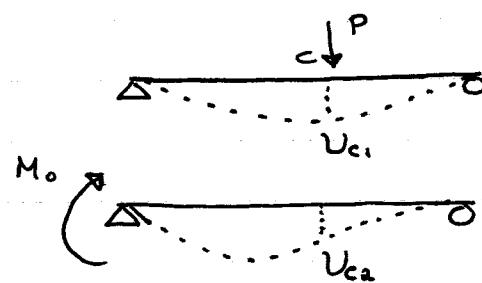
Example :



Determine the deflection @ point C

Given $E = 29 \times 10^3 \text{ ksi}$, $I = 350 \text{ in}^4$

Solution :



$$v_c = v_{c1} + v_{c2}$$

2

(2)

$$U_{c1} = \frac{-PL^3}{48EI}$$

$$U_{c2} = \frac{-M_o x}{6EI L} (x^2 - 3Lx + 2L^2) \Big|_{x=42}$$

$$= -\frac{M_o L^2}{16EI}$$

$$P = 12 \text{ kip} = 12(10^3) \text{ lb}$$

$$M_o = 50 \text{ kip} \cdot ft = 50(10^3) \times 12 \text{ in} = 600(10^3) \text{ lb-in}$$

$$L = 24 \text{ ft} = 24 \times 12 \text{ in} = 288 \text{ in}$$

$$I = 350 \text{ in}^4$$

$$E = 29(10^3) \text{ ksi} = 29(10^6) \text{ psi}$$

$$\therefore U_{c1} = \frac{-PL^3}{48EI} = -\frac{12(10^3)(288)^3}{48(29)(10^6)(350)}$$

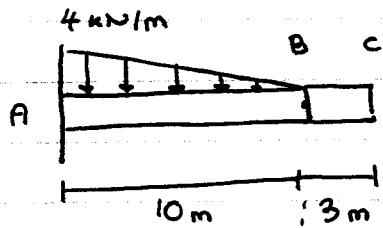
$$= -0.3064 \text{ in}$$

$$U_{c2} = \frac{-M_o L^2}{16EI} = -\frac{600(10^3)(288)^2}{16(29)(10^6)(350)}$$

$$= -0.5884 \text{ in}$$

$$\therefore V_c = U_{c1} + U_{c2} = -0.895 \text{ in}$$

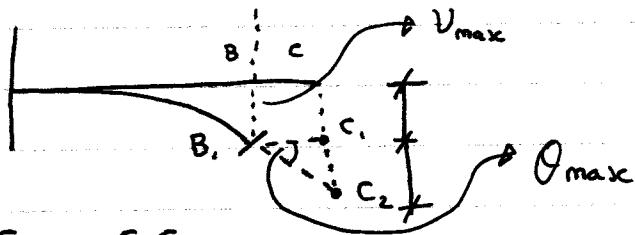
Example:



$$EI = \text{const.}$$

Find the deflection at C.

Solution:



$$V_c = CC_1 + C_1 C_2$$

translation rotation

$$U_{max} = -\frac{w_0 L^4}{30 EI}$$

$$\theta_{max} = -\frac{w_0 L^3}{24 EI}$$

$$C, C_2 = BC \cdot \theta_{max}$$

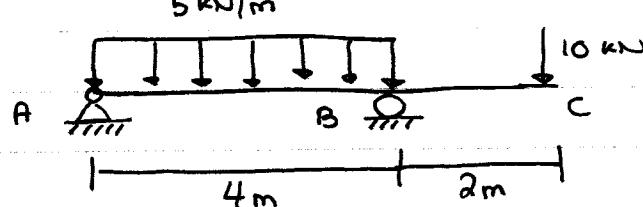
$$\begin{aligned} \therefore U_c &= U_{max} + BC \cdot \theta_{max} \\ &= -\frac{w_0 L^4}{30 EI} + BC \left(-\frac{w_0 L^3}{24 EI} \right) \end{aligned}$$

$$\text{Here, } L = 10 \text{ m} \quad BC = 3 \text{ m}$$

$$w_0 = 4 \text{ kN/m} = 4000 \text{ N/m}$$

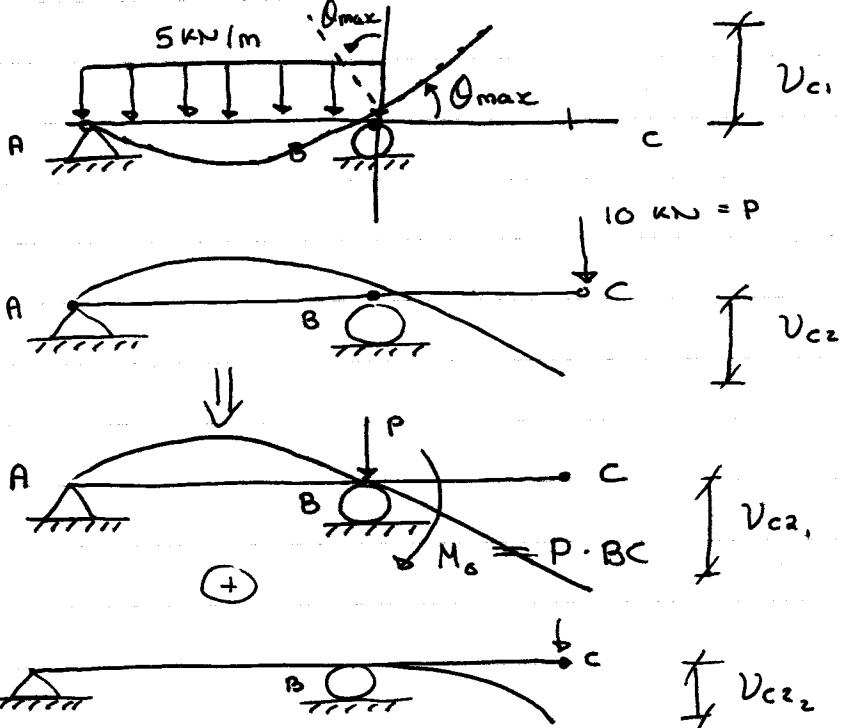
$$\begin{aligned} \therefore U_c &= -\frac{4000 (10)^4}{30 EI} - 3 \times \frac{4000 (10)^3}{24 EI} \\ &= -\frac{1.833 (10^6)}{EI} \end{aligned}$$

Example:



$EI = \text{const}$ → Find the displacement at point C.

Solution:



$$U_{c2} = U_{c21} + U_{c22}$$

rotation bending

$$U_{c1} = BC \cdot \theta_{\max} = 2 \times \frac{\omega_0 L^3}{24EI}$$

$$= 2 \times \frac{5000 (4)^3}{24EI}$$

$$26.67 = \frac{166.67 \times 10^3}{EI}$$



$$U_{c2} = U_{c21} + U_{c22}$$

$$U_{c21} = BC \cdot \theta_i = 2 \left(-\frac{M_0 L}{3EI} \right)$$

$$= -2 \times \frac{P \times BC \times 1}{3EI}$$

$$= -2 \times \frac{10(10^3) \times 2 \times 4}{3EI}$$

$$= -53.3 \times 10^3$$

EI

$$U_{c22} = -\frac{PL^3}{3EI}$$

$$= -\frac{10(10^3)(2)^3}{3EI} \Rightarrow -\frac{26.7 \times 10^3}{EI}$$

$$\therefore U_c = U_{c1} + U_{c2}$$

$$= U_{c1} + U_{c21} + U_{c22}$$

$$= \frac{26.67 \times 10^3}{EI} \cancel{+} \frac{-53.3 \times 10^3}{EI} - \frac{26.7 \times 10^3}{EI}$$

$$= \frac{-53.3 \times 10^3}{EI}$$