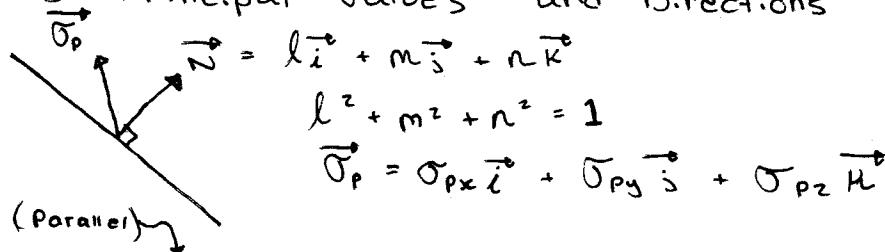


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2.4.3 Principal Values and Directions



$$\vec{\sigma}_p = \sigma_{px}\vec{i} + \sigma_{py}\vec{j} + \sigma_{pz}\vec{k}$$

If $\vec{\sigma}_p \perp \vec{n}$, there is a number σ such that
 $\vec{\sigma}_p = \sigma \vec{n}$

 σ : normal stress \vec{n} : the principal directions

$$\Rightarrow \sigma_{px}\vec{i} + \sigma_{py}\vec{j} + \sigma_{pz}\vec{k} = \sigma(l\vec{i} + m\vec{j} + n\vec{k})$$

$$\Rightarrow \sigma_{px} = \sigma l ; \sigma_{py} = \sigma m ; \sigma_{pz} = \sigma n$$

$$\Rightarrow \begin{cases} \sigma_{xx}l + \sigma_{xy}m + \sigma_{xz}n = \sigma l \\ \sigma_{yx}l + \sigma_{yy}m + \sigma_{yz}n = \sigma m \\ \sigma_{zx}l + \sigma_{zy}m + \sigma_{zz}n = \sigma n \end{cases}$$

$$l^2 + m^2 + n^2 = 1$$

$$\begin{cases} (\sigma_{xx} - \sigma)l + \sigma_{xy}m + \sigma_{xz}n = 0 \\ \sigma_{xy}l + (\sigma_{yy} - \sigma)m + \sigma_{yz}n = 0 \\ \sigma_{xz}l + \sigma_{yz}m + (\sigma_{zz} - \sigma)n = 0 \end{cases}$$

A non-zero solution of (l, m, n)

$$\begin{vmatrix} \sigma_{xx} - \sigma & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} - \sigma & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} - \sigma \end{vmatrix} = 0$$

$$\Rightarrow \sigma^3 - I_1\sigma^2 + I_2\sigma - I_3 = 0$$

$$\text{Here: } I_1 = \sigma_{xx} + \sigma_{yy} + \sigma_{zz}$$

$$I_2 = \sigma_{xx}\sigma_{yy} + \sigma_{yy}\sigma_{zz} + \sigma_{zz}\sigma_{xx} - \sigma_{xy}^2 - \sigma_{yz}^2 - \sigma_{xz}^2$$

$$I_3 = \begin{vmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{vmatrix}$$

 I_1, I_2, I_3 : Invariants of Stress

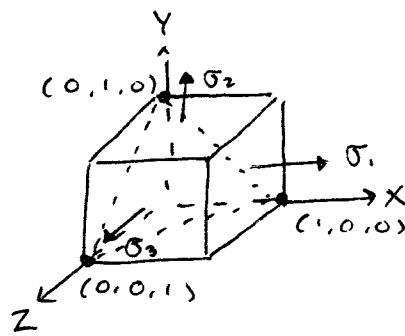
Three roots : $\sigma_1 \geq \sigma_2 \geq \sigma_3$

In the principal axes, the state of stress

$$\begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix}$$

$$\Rightarrow \begin{cases} I_1 = \sigma_1 + \sigma_2 + \sigma_3 \\ I_2 = \sigma_1\sigma_2 + \sigma_2\sigma_3 + \sigma_3\sigma_1 \\ I_3 = \sigma_1\sigma_2\sigma_3 \end{cases}$$

2.4.4 Octahedral Stress



XYZ: the principal directions

Consider the family of planes whose unit normal vectors satisfy :

$$l^2 = m^2 = n^2 = \frac{1}{3}$$

The normal stress :

$$\begin{aligned} \sigma_{\text{out}} &= l^2 \cancel{\sigma_{xx}} + m^2 \cancel{\sigma_{yy}} + n^2 \cancel{\sigma_{zz}} + 2lm \cancel{\sigma_{xy}} + 2mn \cancel{\sigma_{yz}} + 2nl \cancel{\sigma_{zx}} \\ &= \frac{1}{3} (\sigma_1 + \sigma_2 + \sigma_3) = \frac{1}{3} I_1 \\ &= \frac{1}{3} (\sigma_{xx} + \sigma_{yy} + \sigma_{zz}) \end{aligned}$$

$$I_{\text{oct}} = \sqrt{\frac{2}{9} I_1^2 - \frac{1}{3} I_2}$$

2.4.5 Mean and Deviator Stresses

The mean (normal) stress :

$$\sigma_m = \frac{1}{3} (\sigma_{xx} + \sigma_{yy} + \sigma_{zz}) = \frac{1}{3} I_1$$

Define

$$[T_m] = \begin{bmatrix} \sigma_m & 0 & 0 \\ 0 & \sigma_m & 0 \\ 0 & 0 & \sigma_m \end{bmatrix} \quad \begin{array}{c} \text{mean stress} \\ \text{tensor} \end{array}$$

$$[T\Delta] = \begin{bmatrix} \sigma_{xx}-\sigma_m & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy}-\sigma_m & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz}-\sigma_m \end{bmatrix}$$

deviatoric
stress
tensor

$[T_m]$: hydro static stress

$[T_d]$: pure shear stress

Example : The state of stress at a point is given by

$$[T] = \begin{bmatrix} -19.0 & -4.70 & 6.45 \\ -4.70 & 4.60 & 11.8 \\ 6.45 & 11.8 & -8.30 \end{bmatrix} \text{ MPa}$$

Determine the principal stress and their orientation w.r.t. the original coordinate system.

Solution : $I_1 = \sigma_{xx} + \sigma_{yy} + \sigma_{zz}$

$\left. \begin{array}{l} (\text{units aren't necessary}) \\ \text{intermediate step} \end{array} \right\} \quad \begin{aligned} I_2 &= \sigma_{xx}\sigma_{yy} + \sigma_{yy}\sigma_{zz} + \sigma_{zz}\sigma_{xx} - \sigma_{xy}^2 - \sigma_{yz}^2 - \sigma_{xz}^2 \\ &= -170.8125 \text{ MPa}^2 \\ I_3 &= |T| = 2647.522 \text{ MPa}^3 \end{aligned}$

The cubic eq'n:

$$\sigma^3 - I_1 \sigma^2 + I_2 \sigma - I_3 = 0$$

$$\Rightarrow \sigma^3 + 22.7\sigma^2 - 170.8125\sigma - 2647.522 = 0$$

$$\sigma_a = 2s \cos \alpha + \frac{1}{3} I_1$$

$$\sigma_b = 2s \cos(\alpha + 120^\circ) + \frac{1}{3} I_2$$

$$\sigma_c = 2s \cos(\alpha + 240^\circ) + \frac{1}{3} I_3$$

Here $s = \sqrt{\frac{1}{9}I_1^2 - \frac{1}{3}I_2}$

$$\alpha = \frac{1}{3} \cos^{-1} \left(-\frac{Q}{2T} \right)$$

$$Q = \frac{1}{3} I_1 I_2 - I_3 - \frac{2}{27} I_1^3$$

$$T = s^3$$

$$s = \sqrt{\frac{1}{9}(-22.7)^2 - \frac{1}{3}(-170.8125)} = 10.6861$$

$$T = s^3 = 1220.26$$

$$\begin{aligned} Q &= \frac{1}{3}(-22.7)(-170.8125) - (2647.522) - \frac{2}{27}(-22.7) \\ &= -488.590 \end{aligned}$$

$$\therefore \alpha = 1/3 \cos^{-1} \left(-\frac{-488.590}{2 \times 1220.26} \right)$$

$$= 0.466412 \text{ rad} = 26.151^\circ$$

$$\therefore \sigma_a = 2(10.6861) \cos 26.151^\circ + \left(-\frac{22.7}{3}\right) = 11.6178$$

$$\sigma_b = 2(10.6861) \cos (26.151^\circ + 120^\circ) + \left(-\frac{22.7}{3}\right) = -25.3163$$

$$\sigma_c = 2(10.6861) \cos (26.151^\circ + 240^\circ) + \left(-\frac{22.7}{3}\right) = -9.0015$$

$$\therefore \sigma_1 = 11.6178 \text{ MPa}$$

$$\sigma_2 = -9.0015 \text{ MPa}$$

$$\sigma_3 = -25.3163 \text{ MPa}$$

Principal direction for σ_1 :

$$(\sigma_{xx} - \sigma_1)l + \sigma_{xy}m + \sigma_{xz}n = 0$$

$$\sigma_{xy}l + (\sigma_{yy} - \sigma)m + \sigma_{yz}n = 0$$

$$\sigma_{xz}l + \sigma_{yz}m + (\sigma_{zz} - \sigma)n = 0$$

$$\Rightarrow \begin{cases} -30.6178l - 4.70m + 6.45n = 0 \\ -4.70l - 7.0178m + 11.8n = 0 \\ 6.45l + 11.8m - 19.9178n = 0 \end{cases}$$

$$a = \begin{vmatrix} -30.6178 & -4.70 & 6.45 \\ -4.70 & -7.0178 & 11.8 \\ 6.45 & 11.8 & -19.9178 \end{vmatrix} = 0.53914$$

$$b = -\begin{vmatrix} -4.70 & 11.8 \\ 6.45 & -19.9178 \end{vmatrix} = -17.504$$

$$c = \begin{vmatrix} -4.70 & -7.0178 \\ 6.45 & 11.8 \end{vmatrix} = -10.195$$

$$k = \sqrt{a^2 + b^2 + c^2} = \sqrt{(0.53914)^2 + (17.504)^2 + (-10.195)^2} = 20.264$$

$$\therefore l = \frac{a}{k} = \frac{0.53914}{20.264} = 0.02661$$

$$m = \frac{b}{k} = -0.8638$$

$$\left(\text{ver: } l^2 + m^2 + n^2 = 1 \right)$$

$$n = \frac{c}{k} = -0.5031$$

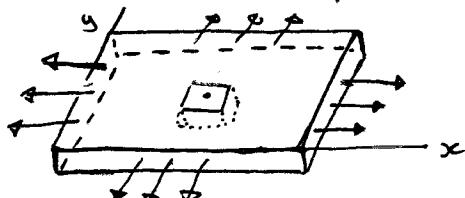
For $\tilde{\sigma}_2$: $(-0.6209, 0.3802, -0.6855)$

For $\tilde{\sigma}_3$: $(-0.7834, -0.3306, 0.5262)$

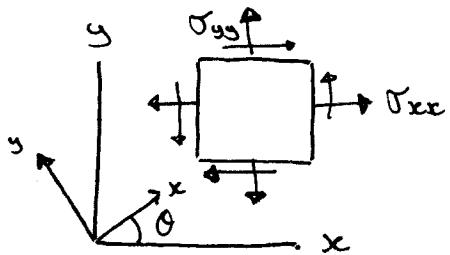
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2.4.6 Plane Stress

Consider a thin plate

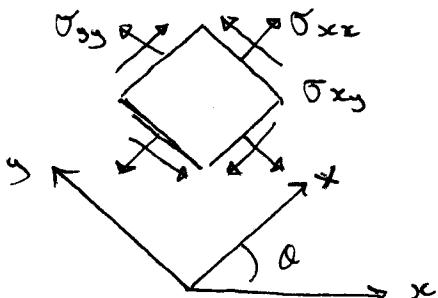


nonzero stress : $\sigma_{xx}, \sigma_{yy}, \sigma_{zz}$ } Plane
others : $\sigma_{zx} = \sigma_{zy} = \sigma_{zz} = 0$ } stress



$$\vec{N}_3 = l_3 \vec{i} + m_3 \vec{j} + n_3 \vec{k}$$

$$l_3 = 0, \quad m_3 = 0, \quad n_3 = 1$$



along the x-axis :

$$\vec{N}_1 = l_1 \vec{i} + m_1 \vec{j} + n_1 \vec{k}$$

$$l_1 = \cos \theta; \quad m_1 = \sin \theta; \quad n_1 = 0$$

$$\vec{N}_2 = l_2 \vec{i} + m_2 \vec{j} + n_2 \vec{k}$$

$$l_2 = -\sin \theta; \quad m_2 = \cos \theta; \quad n_2 = 0$$

$$\sigma_{xx} = l^2 \sigma_{xx} + m^2 \sigma_{yy} + n^2 \cancel{\sigma_{zz}} \dots$$

$$\dots + 2l \cdot m \cdot \sigma_{xy} + 2m \cdot n \cdot \cancel{\sigma_{yz}} \dots$$

$$\dots + 2n \cdot l \cdot \cancel{\sigma_{xz}}$$

$$\Rightarrow \sigma_{xx} = \sigma_{xx} \cos^2 \theta + \sigma_{yy} \sin^2 \theta + 2\sigma_{xy} \sin \theta \cos \theta$$

$$\Rightarrow * \sigma_{xx} = \sigma_{avg} + \frac{\sigma_{xx} - \sigma_{yy}}{2} \cos 2\theta + \sigma_{xy} \sin 2\theta$$

$$\sigma_{avg} = \frac{\sigma_{xx} + \sigma_{yy}}{2} = \frac{l_1}{2}$$

$$\sigma_{yy} = \sigma_{avg} - \frac{\sigma_{xx} - \sigma_{yy}}{2} \cos 2\theta - \sigma_{xy} \sin 2\theta$$

$$(*) \sigma_{xy} = -\frac{\sigma_{xx} - \sigma_{yy}}{2} \sin 2\theta + \sigma_{xy} \cos 2\theta$$

Max or min. normal stress occurs when

$$\boxed{\sigma_{xy} = 0}$$

Principal stresses

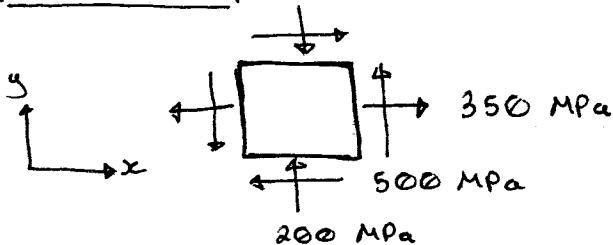
$$\Rightarrow \tan 2\theta_p = \frac{2\sigma_{xy}}{\sigma_{xx} - \sigma_{yy}}$$

the principal stresses :

$$\sigma_1 = \sigma_{avg} + R, \quad \sigma_2 = \sigma_{avg} - R$$

$$\text{Here, } R = \sqrt{\left(\frac{\sigma_{xx} - \sigma_{yy}}{2}\right)^2 + \sigma_{xy}^2}$$

Example



Find the principal stresses and the orientations.

$$\text{Solution: } \sigma_{xx} = 350 \quad \sigma_{yy} = -200 \\ \sigma_{xy} = 500$$

$$\sigma_{avg} = \frac{\sigma_{xx} + \sigma_{yy}}{2} = \frac{350 - 200}{2} = 75$$

$$R = \sqrt{\left(\frac{\sigma_{xx} - \sigma_{yy}}{2}\right)^2 + \sigma_{xy}^2}$$

$$\Rightarrow \sqrt{\left(\frac{350 + 200}{2}\right)^2 + 500^2} \Rightarrow 570.64$$

$$\therefore \sigma_1 = \sigma_{avg} + R = 75 + 570.64 = 645.64$$

$$\sigma_2 = \sigma_{avg} - R = 75 - 570.64 = -495.36$$

$$\tan 2\theta_p = \frac{2\sigma_{xy}}{\sigma_{xx} - \sigma_{yy}} = \frac{2(500)}{350 - 200} = 1.82$$

$$2\theta_p = 61.19^\circ, \quad 61.19^\circ + 180^\circ$$

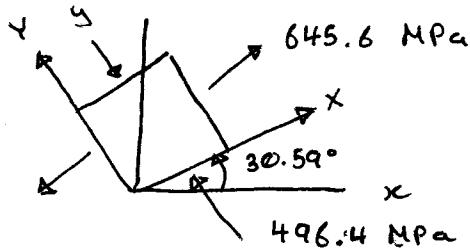
$$\theta_p = 30.59^\circ, \quad 120.59^\circ$$

Let $\theta = 30.59^\circ$

$$\sigma_{xx} = \sigma_{avg} + \frac{\sigma_{xx} - \sigma_{yy}}{2} \cos 2\theta + \sigma_{xy} \sin 2\theta$$

$$= 75 + \frac{350 + 200}{2} \cos 61.19^\circ + 500 \sin 61.19^\circ$$

$$= 645.64$$



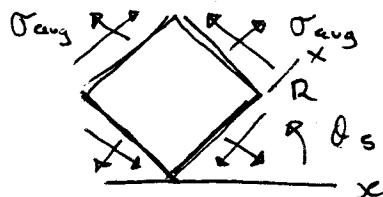
Max in-plane shear stress

$$\frac{d\sigma_{xy}}{d\theta} = 0 \Rightarrow -\frac{\sigma_{xx} - \sigma_{yy}}{2}(2\cos 2\theta) - \sigma_{xy}(2\sin 2\theta) = 0$$

$$\Rightarrow \boxed{\tan 2\theta_s = -\frac{\sigma_{xx} - \sigma_{yy}}{2\sigma_{xy}}}$$

and $\sigma_{xy, \text{max}} = R$
in-plane

In the directions of the max in-plane shear stress,
the normal stress is σ_{avg}

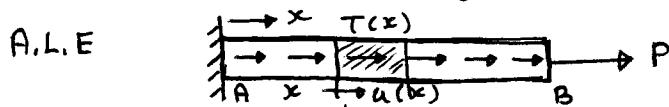


2.4.7

READING

2.5 Differential equations of motion of a deformable body

1° 1D stress analysis - axial member



the displacement at B?
[$\delta(x)$]

$$A\delta(x) \xleftarrow{x} \boxed{\quad} \xrightarrow{x+dx} (\delta(x) + d\delta) A$$

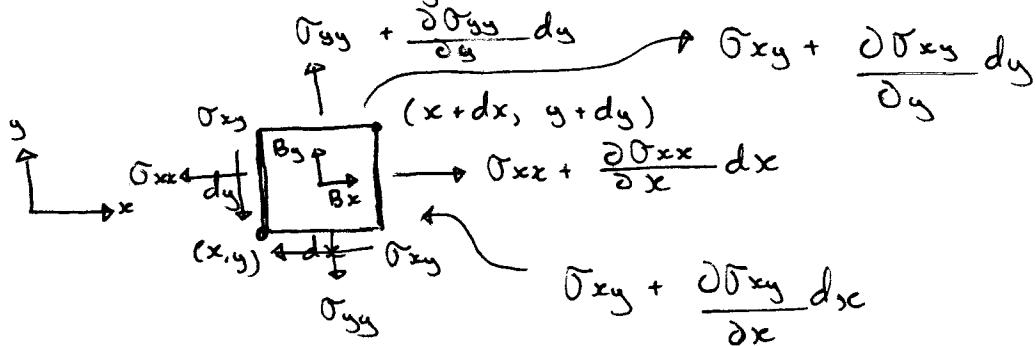
$$\sum F_x = 0 :$$

$$\begin{aligned} T(x) dx + (\sigma(x) + d\sigma) A - A \sigma(x) &= 0 \\ \Rightarrow A d\sigma + T(x) dx &= 0 \\ \Rightarrow \boxed{\frac{d\sigma}{dx} + \frac{T(x)}{A} = 0} \end{aligned}$$

Define: $B_x = \frac{T(x)}{A}$ (force / volume)

$$\boxed{\frac{d\sigma}{dx} + B_x = 0}$$

2° 2D stress analysis:



$$\sum F_x = 0 :$$

$$\begin{aligned} -\sigma_{xx} dy + (\sigma_{xx} + \frac{\partial \sigma_{xx}}{\partial x} dx) dy &\dots \\ \dots -\sigma_{xy} dx + (\sigma_{xy} + \frac{\partial \sigma_{xy}}{\partial y} dy) dx &\dots \\ \dots + B_x dx dy &= 0 \\ \Rightarrow \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} + B_x &= 0 \end{aligned}$$

$$\sum F_y :$$

$$\frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + B_y = 0$$

3° 3D

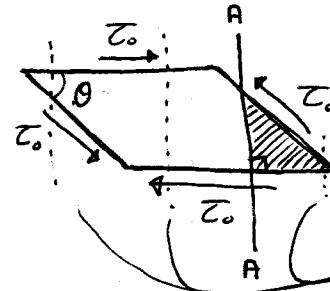
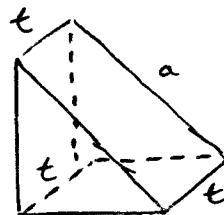
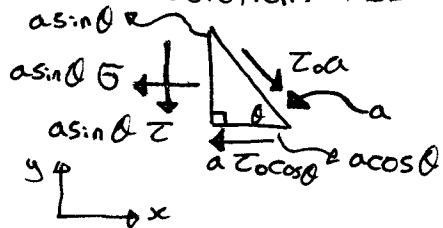
$$\left\{ \begin{array}{l} \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} + \frac{\partial \sigma_{xz}}{\partial z} + B_x = 0 \\ \frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{yz}}{\partial z} + B_y = 0 \\ \frac{\partial \sigma_{xz}}{\partial x} + \frac{\partial \sigma_{yz}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z} + B_z = 0 \end{array} \right.$$



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Example: A thin plate is subjected to uniform shear stress. Find the stress vector on the cutting plane A-A

Solution: FBD



Same results for diff. cutting plane.

$$\sum F_x = 0 :$$

$$-\sigma \sin \theta - Z_0 \cos \theta - Z_0 \cos \theta = 0$$

$$\Rightarrow \sigma = -2Z_0 \frac{\cos \theta}{\sin \theta} = -2Z_0 \cot \theta \quad (\text{normal stress})$$

$$\sum F_y = 0 :$$

$$-Z_0 \sin \theta + Z_0 \sin \theta = 0$$

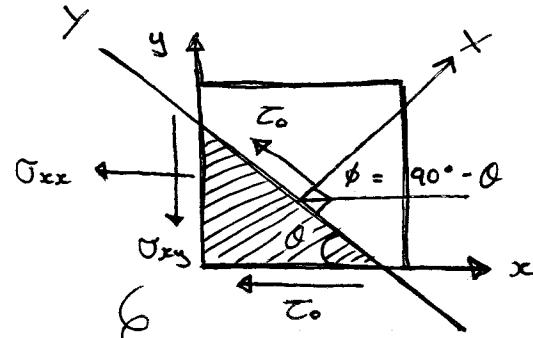
$$\Rightarrow \tau = Z_0 \quad (\text{shear stress})$$

Method 2 :

(Putting small triangle into differential element)

State of Stress

$$\left\{ \begin{array}{l} \sigma_{xx} = \text{unknown} \\ \sigma_{yy} = 0 \\ \sigma_{xy} = \text{unknown} \end{array} \right.$$



By inspection, you can see $\sigma_{xy} = Z_0$ right away

Stress transformation

$$\sigma_{xx} = 0 = \sigma_{xx} \cos^2(90^\circ - \theta) + \sigma_{yy} \sin^2(90^\circ - \theta) + \sigma_{xy} \sin(90^\circ - \theta) \cos(90^\circ - \theta)$$

$$\Rightarrow \sigma_{xx} \sin^2 \theta + 2\sigma_{xy} \sin \theta \cos \theta = 0 \quad (1)$$

$$\sigma_{xy} = Z_0 = -\left(\frac{\sigma_{xx} - \sigma_{yy}}{2}\right) \sin 2(90^\circ - \theta) + \sigma_{xy} \cos 2(90^\circ - \theta)$$

$$Z_0 = -\left(\frac{\sigma_{xx}}{2}\right) \sin 2\theta - \sigma_{xy} \cos 2\theta \quad (2)$$

$$\Rightarrow \sigma_{xx} = -2Z_0 \cot \theta \quad \Rightarrow \sigma_{xy} = Z_0$$

Example :

$$[\tau] = \begin{bmatrix} 5 & 0 & 0 \\ 0 & -6 & -12 \\ 0 & -12 & 1 \end{bmatrix} \text{ (MPa)}$$

Determine

- the principal stress $(10, 5, -15)$
- the principal directions

Solution : $\sigma_{xx} = 5 \quad \sigma_{xy} = \sigma_{xz} = 0$

$$\sigma_{yx} = 0 \quad \sigma_{yy} = -6 \quad \sigma_{yz} = -12$$

$$\sigma_{zx} = 0 \quad \sigma_{zy} = -12 \quad \sigma_{zz} = 1$$

and $I_1 = \sigma_{xx} + \sigma_{yy} + \sigma_{zz} = 5 - 6 + 1 = 0$

$$I_2 = \sigma_{xx}\sigma_{yy} + \sigma_{yy}\sigma_{zz} + \sigma_{zz}\sigma_{xx} - \sigma_{xy}^2 - \sigma_{yz}^2 - \sigma_{zx}^2$$

$$\Rightarrow -175$$

$$I_3 = \begin{vmatrix} 5 & 0 & 0 \\ 0 & -6 & -12 \\ 0 & -12 & 1 \end{vmatrix} = 5 \begin{vmatrix} -6 & -12 \\ -12 & 1 \end{vmatrix} = -750$$

$$\Rightarrow \sigma^3 - I_1\sigma^2 + I_2\sigma - I_3$$

$$\sigma^3 - 175\sigma + 750 = 0$$

$\sigma = 5$ is a root (by inspection)

$\sigma = 10$ and $\sigma = -15$ are the rest of the roots

$$\therefore \sigma_1 = 10, \quad \sigma_2 = 5, \quad \sigma_3 = -15$$

$$\frac{\sigma^3 - 175\sigma + 750}{\sigma - 5} = \sigma^2 + 5\sigma - 150 = 0$$